

## Hydropower Plants: Generating and Pumping Units Solved Problems: Series 6

## 1 NON-DIMENSIONAL NUMBERS FOR A TURBINE MODEL TEST

When performing model testing, an accurate calculation of the non-dimensional numbers is of key importance for the operation of hydraulic turbines and the transposition of the efficiency from the model scale to the prototype scale. Three non-dimensional parameters are particularly important: the IEC speed factor  $n_{ED}$ , the IEC discharge factor  $Q_{ED}$ , and the Thoma number  $\sigma$ .

1) When performing the model test of a turbine, which important quantities need to be monitored for comparing the efficiency and the operating condition of the prototype scale with that of the reduced scale turbine?

Also, which component/parameter must be exactly the same in the prototype and model scales?

It is important to monitor the available head H, the discharge Q, the turbine rotational frequency n, and the torque T, as they are essential for knowing the efficiency of the machine and compare the characteristics to the prototype.

For comparison purposes, the geometry of the turbine must be exactly the same as the prototype.

2) Express the non-dimensional parameters  $n_{ED}$  and  $Q_{ED}$  as functions of the reference diameter  $D_{ref}$ , the discharge Q, the turbine specific energy E, and the rotational frequency n [s<sup>-1</sup>].

$$n_{ED} = \frac{nD_{ref}}{\sqrt{E}}$$

$$Q_{ED} = \frac{Q}{D_{vot}^2 \sqrt{E}}$$

3) Express the specific speed  $\nu$  and IEC specific speed  $n_{QE}$  as functions of  $Q_{ED}$  and  $n_{ED}$ .

$$v = 2^{\frac{1}{4}} \sqrt{\pi Q_{ED}} n_{ED}$$

$$n_{QE} = \sqrt{Q_{ED}} n_{ED}$$

For questions 4) to 6), use the information about the installation of the turbine and the setting level  $h_s$  which are depicted in Figure 1.

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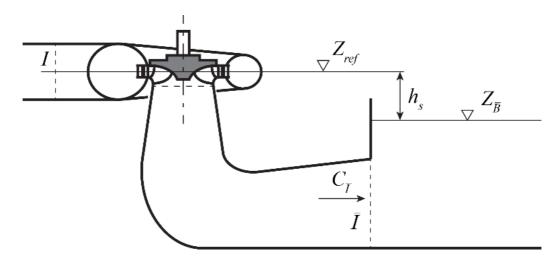


Figure 1: Machine setting level

4) Express the Net Positive Suction Specific Energy (*NPSE*) as a function of  $gH_{\bar{I}}$ ,  $Z_{ref}$ , and the saturated pressure  $p_{\nu}$ .

$$NPSE = gH_{\overline{I}} - \frac{p_{v}}{\rho} - gZ_{ref}$$

5) Express the Thoma number  $\sigma$  defined by  $\frac{NPSE}{E}$ , using the setting level  $h_s = Z_{ref} - Z_{\overline{B}}$ , the flow velocity  $C_{\overline{I}}$ , the saturated pressure  $p_v$  and the atmosphere pressure  $p_a$ . Take the singular losses term  $gH_{r\overline{I}+\overline{B}}$  into account, knowing that the draft tube outlet can be considered as a water outflow with  $K_v = 1$ .

$$\sigma = \frac{NPSE}{E} = \frac{gH_{\overline{I}} - \frac{p_{v}}{\rho} - gZ_{ref}}{E} = \frac{\frac{p_{a}}{\rho} + gZ_{\overline{B}} + \frac{1}{2}C_{\overline{I}}^{2} - \frac{p_{v}}{\rho} - gZ_{ref}}{E} = \frac{\frac{p_{a} - p_{v}}{\rho} - gh_{s} + \frac{1}{2}C_{\overline{I}}^{2}}{E}$$

6) Describe what is the risk of operating the plant at a low Thoma number.

At low Thoma number, a turbine is subject to cavitation which occasionally gets unstable, and often causes serious problems for hydraulic systems such as erosion, fatigue, efficiency loss, resonance and power swings.

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